

# Autodesk® Simulation for Designers, not Doctors! (No Phd Required) SM3257-L

**Kevin Marchant**

Application Engineer (D3 Technologies)

**Dave Graves**

Sr. Technical Specialist (Autodesk)

## Cartesian coordinates [\[edit\]](#)

With the velocity vector expanded as  $\mathbf{v} = (u, v, w)$ , we may write the vector equation explicitly,

$$\rho \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = -\frac{\partial p}{\partial x} + \mu \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) + \rho g_x,$$

$$\rho \left( \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right) + \rho g_y$$

$$\rho \left( \frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right) + \rho g_z.$$

Note that gravity has been accounted for as a body force, and the values of  $g_x$ ,  $g_y$ ,  $g_z$  will depend on the orientation of gravity with respect to the chosen set of coordinates.

The continuity equation reads:

$$\frac{\partial \rho}{\partial t} + \frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = 0.$$

When the flow is incompressible,  $\rho$  does not change for any fluid parcel, and its [material derivative](#) vanishes:  $\frac{D\rho}{Dt} = 0$ . The continuity equation is reduced to:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0.$$

The velocity components (the dependent variables to be solved for) are typically named  $u$ ,  $v$ ,  $w$ . This system of four equations comprises the most commonly used and studied form. Though comparatively more compact than other representations, this is still a [nonlinear system of partial differential equations](#) for which solutions are difficult to obtain.

## Cylindrical coordinates [\[edit\]](#)

A change of variables on the Cartesian equations will yield<sup>[15]</sup> the following momentum equations for  $r$ ,  $\phi$ , and  $z$ :

$$r: \rho \left( \frac{\partial u_r}{\partial t} + u_r \frac{\partial u_r}{\partial r} + \frac{u_\phi}{r} \frac{\partial u_r}{\partial \phi} + u_z \frac{\partial u_r}{\partial z} - \frac{u_\phi^2}{r} \right) = -\frac{\partial p}{\partial r} + \mu \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial u_r}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_r}{\partial \phi^2} + \frac{\partial^2 u_r}{\partial z^2} - \frac{u_r}{r^2} - \frac{2}{r^2} \frac{\partial u_\phi}{\partial \phi} \right] + \rho g_r$$

$$\phi: \rho \left( \frac{\partial u_\phi}{\partial t} + u_r \frac{\partial u_\phi}{\partial r} + \frac{u_\phi}{r} \frac{\partial u_\phi}{\partial \phi} + u_z \frac{\partial u_\phi}{\partial z} + \frac{u_r u_\phi}{r} \right) = -\frac{1}{r} \frac{\partial p}{\partial \phi} + \mu \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial u_\phi}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_\phi}{\partial \phi^2} + \frac{\partial^2 u_\phi}{\partial z^2} + \frac{2}{r^2} \frac{\partial u_r}{\partial \phi} - \frac{u_\phi}{r^2} \right] + \rho g_\phi$$

$$z: \rho \left( \frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\phi}{r} \frac{\partial u_z}{\partial \phi} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial u_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \phi^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho g_z.$$

# Who -

- **Your Instructors**

- **Kevin Marchant**

- AE D3 Technologies

- **Dave Graves**

- Sr. Technical Specialist (Autodesk)

- **Lab Assistants**

- **Wasim Younis**

- Symetri UK

- **James Neville**

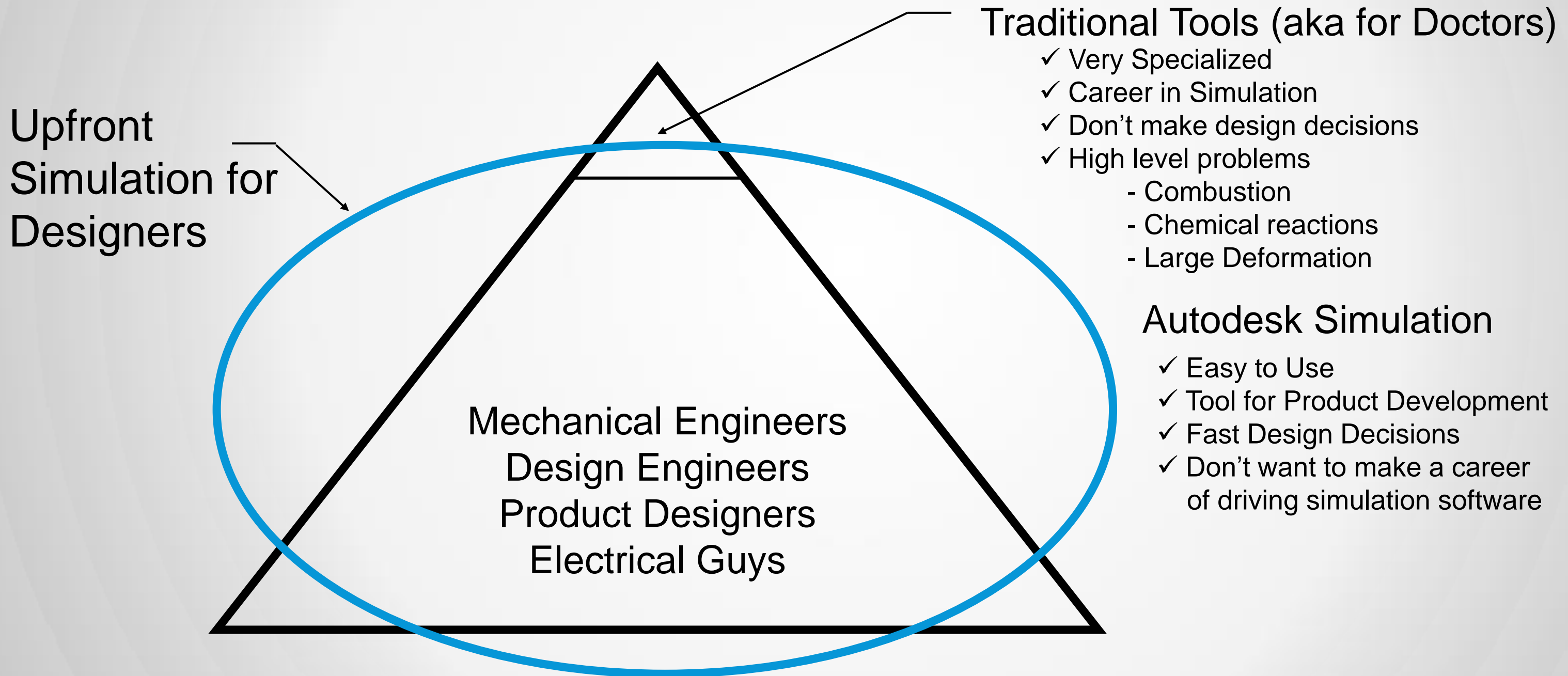
- Technical Specialist (Autodesk)

- **Peggy Menzies, Phd**

- CFD Quality Lead (Autodesk)

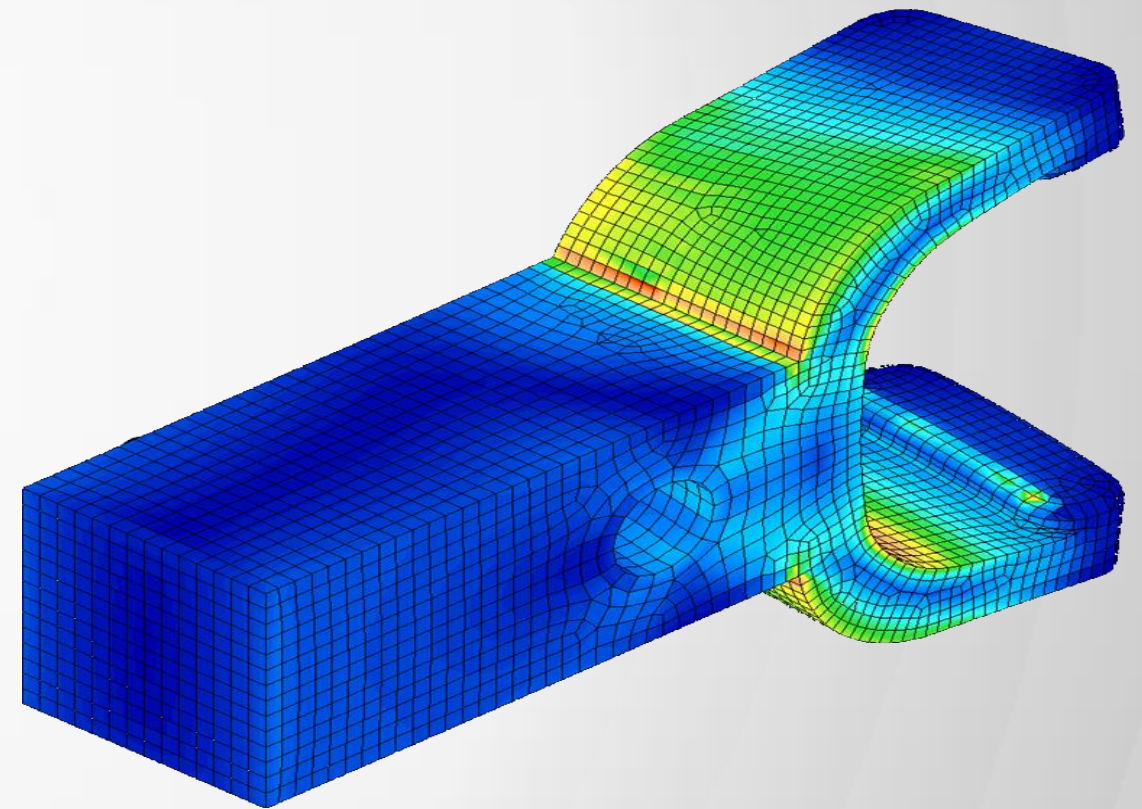
# Upfront Simulation in Engineering

## Simulation for the Designers



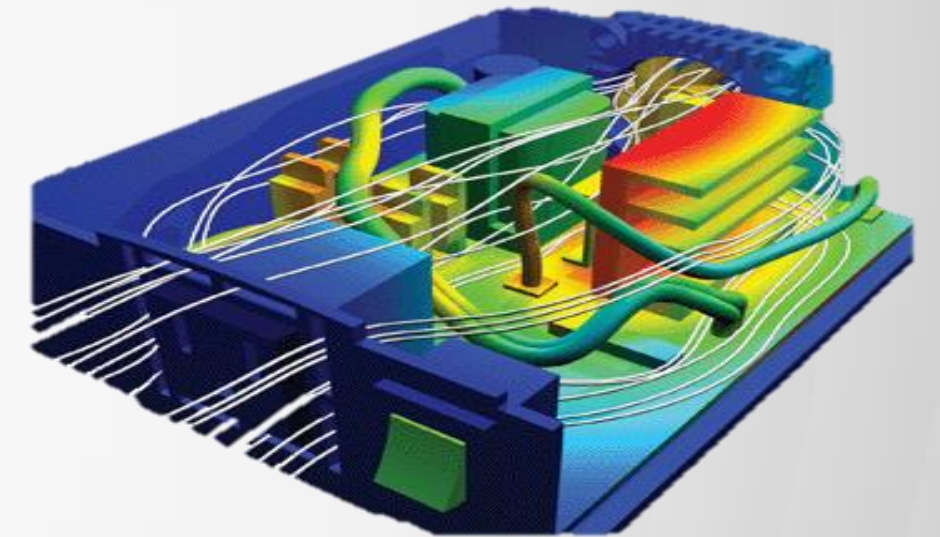
# What is FEA Analysis?

- A method to evaluate the physical performance of a design before it is built
- FEA = Finite Element Analysis
- Finite = Limited, discrete
- Elements = Small block which results can be solved for



# What is CFD Analysis?

- The modeling of fluid flow as it interacts with surrounding surfaces
- CFD = **C**omputational **F**luid **D**ynamics
- Fluid = Liquids and gases
- Dynamics = Movement
- CFD tools today also have the capability to model the effects of **temperature**
- Flow and Thermal Analysis
- Many Industries across **MFG, AEC, ENI**



# Let's get started: (Key learning objectives)

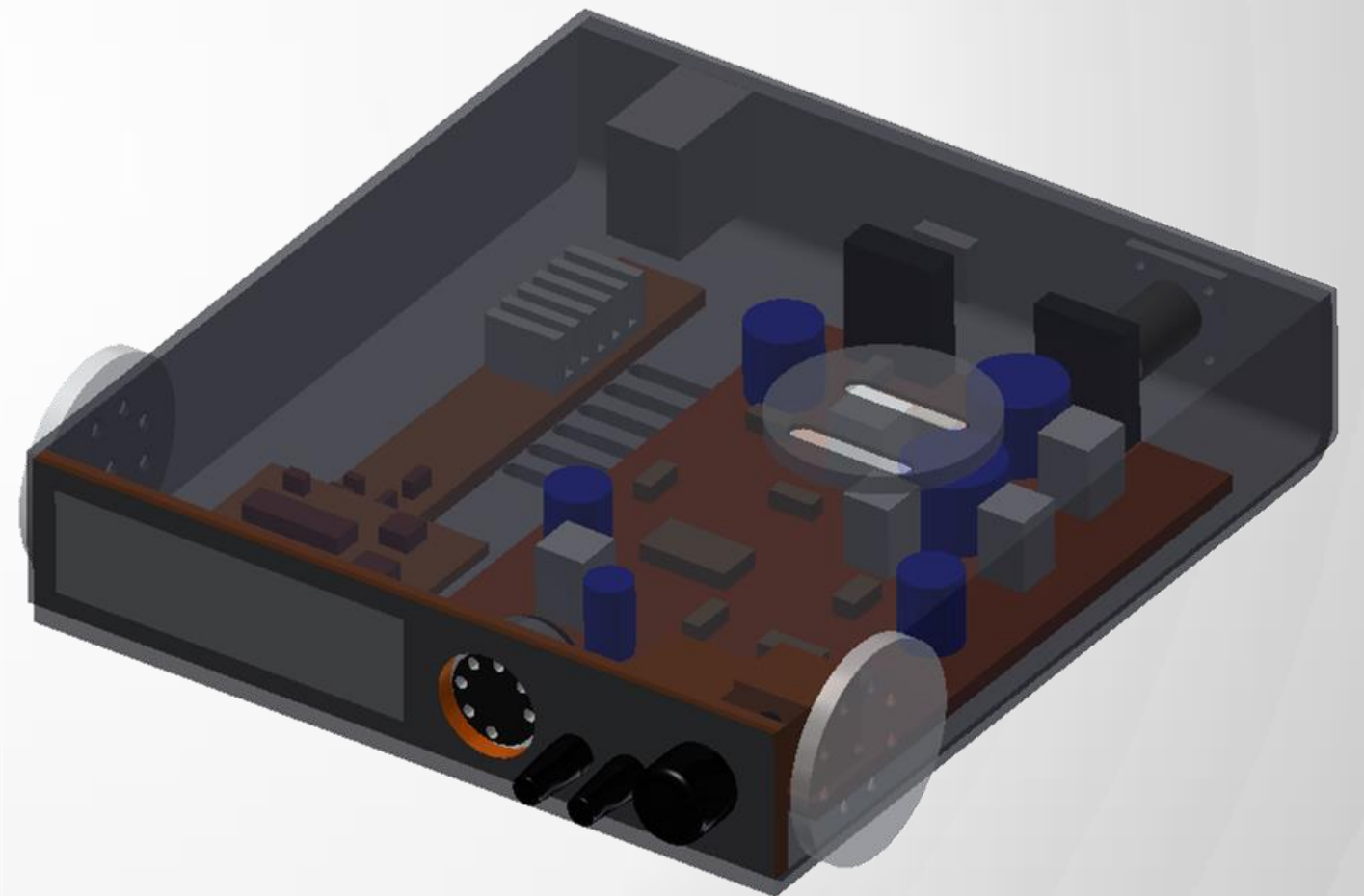
At the end of this class, you will be able to:

- Create an analysis model from Inventor
- Set up and simulate in CFD
- Process thermal results in CFD
- Evaluate and compare a design change



# Open the Demo Assembly in Inventor

- Navigate to \*\\SM3257-L\Demonstration\Inventor Geometry
- Open “CB.iam”



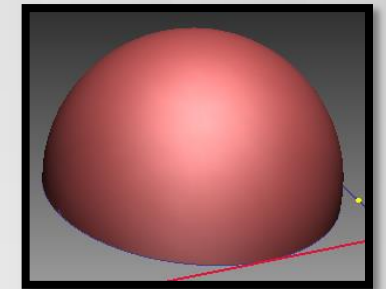


# Basics in CFD

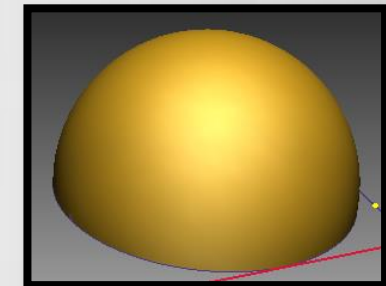
- Complete launch → click OK
- Adjust Geometry → Select Merge
- Practice Navigation and Selection:

Display Action	ADSK CFD Mouse Command
Wheel Zoom	Scroll
Rotate	Shift + MMB
Pan	MMB
Select/deselect	LMB
Rubberband select	LMB drag
Blank/hide	Ctrl + MMB
Show all	Ctrl + MMB off model

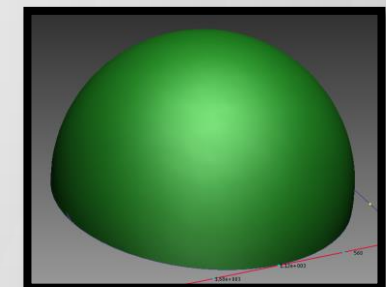
Selected



About to Deselect



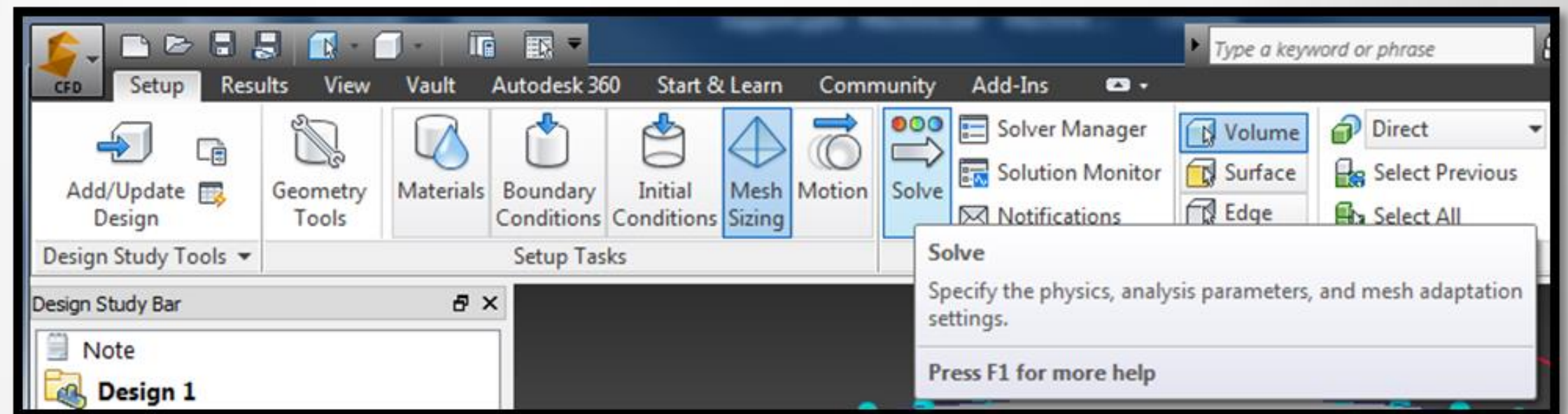
Ready to Select



# Steps

- Materials
- Conditions
- Mesh
- Solve
- Review Results

Work left to right ->



**Thank you!**

# Questions?



